Chiller Plant Energy Optimization Modeling and Decision Analysis

Chun-Wei Chen, Ssu-Yu Chen, Chun-Chang Li, Chen-Yu Lin

Taiwan Instrument Research Institute, NARLabs, 20, R&D Rd. VI, Hsinchu Science Park, Hsinchu, 30076, Taiwan, R.O.C.

Abstract: The objective of this study was to identify a method that would enable a chiller plant to achieve the lowest possible operation energy consumption. Using mathematical programming, this study adopted the actual cooling capacity required by an air conditioning system as a basis for considering the operating performance and condition of each chiller plant component as well as the relationship among components to construct a decision analysis model for chiller plant energy optimization. The problem-solving performances of various algorithms were investigated. Finally, the model architecture and the algorithm that showed most satisfactory performance were employed to establish a decision analysis model for chiller plant energy optimization, which can serve as an objective reference for enabling chiller plant designers and operators to perform optimized equipment control.

Keywords: central air conditioning system, chiller plant, energy optimization, mathematical programming, decision support system

1. Introduction

The pursuit of a comfortable and convenient life has resulted in rapid technological development and increasing energy consumption. In addition to continually increasing energy costs due to energy overexploitation worldwide, the emphasis countries have placed on energy conservation and environmental protection because of global warming has also led to an inevitable increase in cost of electricity.

Taiwan is located in subtropical latitudes, characterized by hot and humid summers. Therefore, air conditioning (AC) systems (referred to as central AC systems consisting of a chiller, chilled water pump, condenser water pump, cooling tower, air handling unit, indoor air cooler, and related pipe lines and valves) are commonly used to maintain a comfortable environment in the indoor spaces of residential buildings. However, this results in large amount of energy consumption (primarily electric energy). The energy consumed by AC systems accounts for 40%–65% of the total energy consumption of an entire building. In particular, chillers are the primary source of power for refrigeration in central AC systems, and they require a large amount of electricity to produce cooled air for an entire building. Generally, the electricity required by a chiller is 50%–65% of the electricity consumed by an entire AC system. Therefore, studies on improving the energy efficiency of AC systems have primarily aimed to achieve optimal operating efficiency of chillers, chilled water pump, condenser water pump, and cooling tower has rarely been considered. Because of the mutual influence of these components, only optimizing the power consumption of certain components may cause increases in the power consumption of other components. Thus, such a method can only realize local optimization and is unable to achieve optimal operating efficiency of a chiller plant system.

Chiller plants (central AC waterside systems) are also referred to as central AC chilled water systems, which play the role of refrigeration, chilled water distribution, and heat removal. It consists of a chiller unit for refrigeration, a chilled water pump for chilled water delivery, a condenser water pump for heat transmission, and a cooling tower for heat removal. Its function is similar to that of the human heart. Because the chiller plant system is responsible for the cooling source (chilled water) for an entire building or factory, its power consumption is considerable and typically accounts for 60%–75% of the total power consumption of the AC system and 25%–45% of the total power consumption of a building. Thus, optimizing the operating efficiency

of a chiller plant can effectively reduce its power consumption and thereby substantially reduce energy costs.

The operation of chiller plant components is typically set and modulated according to human experience or proportional—integral—derivative (PID) controllers based on pressure and temperature. However, these methods are not scientific and systematic and lack the logical basis of optimization, resulting in inaccurate chiller plant operation and excessive energy consumption. Therefore, the objective of this study was to investigate methods for effectively optimizing the operating models and parameters of the components of chiller plants, thereby enhancing overall operating efficiency and reducing energy cost.

2. Traditional Control of Chiller Plants

2.1 Judgment according to human experience

Experience-based control is generally inaccurate and unscientific. Specifically, the components of a chiller plant are set and modulated by on-site operators based on the weather, the use status of indoor space, and their personal experience. However, operators often tend to overly increase the operating capacity of the chiller plant to ensure sufficient cooling capacity for the building in question. In other words, the control of chiller plants depends solely on the subjective judgment of operators and lacks objective scientific control procedures. Thus, the operation of chiller plants is unreliable, unstable, and not energy efficient.

2.2 Control based on temperature and pressure

The control methods that have been most extensively applied to chiller plants are PID controllers and direct digital control that incorporates a PID controller. Both of these control methods use changes in temperature and pressure as the basis for setting and modulating the operating status of various components of a chiller plant.

However, neither PID control nor direct digital control are capable of achieving optimal chiller plant operation for two main reasons. First, the control models that use PID as the basic control logic adopt only independent control variables, such as temperature and pressure, to set and modulate the operating status of chiller plant components. However, independent control variables are not directly related to energy efficiency optimization. These models thus fail to consider the relationship among components from the perspective of energy efficiency optimization and are unable to effectively coordinate and integrate the operation of various components. Thus, chiller plants often have low-efficiency operation. Second, a typical PID feedback control uses the binary values of setpoint and feedback signal as well as proportional, integral, and derivative terms to calculate a proportional value before using the proportional value to reset and modulate the operating status of the controlled equipment. Because calculations of the proportional value do not consider the operating performance of the equipment in question, the optimal proportional value for the operating performance optimization of a chiller plant cannot be easily obtained. This may cause the chiller plant to operate under a nonoptimized state and lead to unnecessary energy consumption. Notably, the operating efficiency of the chiller plant may increase if the setpoints of all components are being continually adjusted (i.e., resetting the PID parameters), and this can lead to unstable operation.

3. Decision Analysis System for Chiller Plant Energy Optimization

Studies on the optimization of AC system operation have revealed two trends. One is using optimization methods, such as artificial neural networks, fuzzy reasoning, grey systems, operation research, system simulation, and dynamic programming, to explore and evaluate the optimal operation parameters for specific systems such as chiller, chilled water side, cooling water side, and airside systems. The other trend is investigating the performance of various optimization algorithms on determining optimal operation values. Both of these research approaches attempt to develop a method for realizing optimal AC system operation and thereby achieve the goal of energy conservation.

The operation of component equipment of chiller plants is typically set and modulated based on human experience or using PID controllers based on pressure and temperature. Because these methods are not scientific or systematic and lack the logical basis of optimization, chiller plants may operate ineffectively and consume excessive amounts of energy. Accordingly, this study referred to studies on AC system optimization to construct an optimization model for the decision-making of chiller plant operation and solve the optimization problem, thereby increasing operation efficiency and avoiding energy waste. Theoretically, if the operation of a chiller plant can be optimized, then resources can be optimally allocated and there will be no need for additional energy conservation. The steps and methods for establishing the decision analysis model for chiller plant energy

optimization are as follows:

- Step 1: A monitoring system (or related apparatus) is used to measure and collect the operation data of various component equipment of the chiller plant in question, and the data are compiled.
- Step 2: A polynomial regression is performed to determine the operation performance function for each chiller plant component.
- Step 3: Mathematical programming is conducted to establish decision analysis model for chiller plant energy optimization.
- Step 4: Simulated annealing (SA), genetic algorithm (GA), particle swarm optimization (PSO), and harmony search (HS) are used to identify the decision variables for the decision analysis model for chiller plant energy optimization. The algorithms are then compared in terms of their performance.
- Step 5: A decision support system is employed to establish the decision analysis model for chiller plant energy optimization.

In summary, this study developed a decision analysis (support) model for chiller plant energy optimization that enables rapid model construction and analysis and can serve as an objective basis for equipment designers and operation management personnel to perform optimized control of chiller plant components. The operation of chiller plant components can be optimized based on the optimization parameters, and the chiller plant operation can thus become both energy-saving and efficient. The main function of the developed system was to help users simulate the optimal start—stop model and operating load for all chiller plant components under various cold energy (unit: refrigeration ton [RT]) requirements and calculate the total power after optimization. The results can serve as an objective and reliable basis for users to improve the operation of chiller plants.

4. Case Test and Analysis

4.1 Specification of the case system

An actual chiller plant was used to test the performance of the energy optimization decision analysis system developed in this study. The case system was a small chiller plant, the capacity of which was 600 RT. It consisted of two centrifugal chillers, two centrifugal coupled primary chilled water pumps, two centrifugal coupled secondary chilled water pumps, two centrifugal coupled condenser water pumps, and two centrifugal cooling tower fans connected in parallel. The first and second chillers were separately provided with chilled water and cooled water by the first and second primary chilled water pump and condenser water pump, respectively. The temperature of the chilled water was 7°C, and that of the cooled water was 30°C. The equipment specifications and operating conditions are listed in Table 1.

4.2 Energy optimization decision analysis (support) system for the case system

Based on the collected operation data of the case system, a polynomial regression was performed to determine the operation performance function of each chiller plant component. Based on the previously established architecture and problem-solving method for the decision analysis model for chiller plant energy optimization, this study used the performance function and the actual specification and operating conditions (requirements) of the case system and employed Borland C++ Builder to write and edit the computing program and interface of the energy optimization decision analysis (support) system. Figure 1 shows the operating interface of the decision analysis (support) system developed in this study.

The area in red labeled "number 1" comprises operation buttons for users to start and stop calculations and save the results.

The area in red labeled "number 2" shows the parameters, including algorithm parameters and related requirements (constraints); values are input by users.

The area in red labeled "number 3" presents the constraints. The system calculates the constraint values according to the parameters. Subsequently, all results should satisfy the constraints.

The area in red labeled "number 4" indicates the record history, which lists all of the calculation results that can serve as a reference for subsequent problem solving.

The area in red labeled "number 5" is a convergence diagram produced from the system calculation. The horizontal axis is the number of calculations, and the vertical axis is the amount of energy consumed.

Table 1. Specification of the case chiller plant

| | Device | Capacity | Water pressure | OLR Min. | OLR _{Max} . | | | | |
|----------|-------------------------------|-------------------------|-----------------------------|--|----------------------|--|--|--|--|
| . , | niller | | | | | | | | |
| | CH ₁ | 300(RT) | - | 40(%) | 100(%) | | | | |
| (| CH ₂ | 300(RT) | - | 40(%) | 100(%) | | | | |
| (II) Pr | imary Chilled Water Pump(| Constant Speed Pump) | | | | | | | |
|] | PCP_1 | 750(GPM) | 3.5(KG/cm ²) | 100(%) | 100(%) | | | | |
| 1 | PCP ₂ | 750(GPM) | $3.5(KG/cm^2)$ | 100(%) | 100(%) | | | | |
| (III) Se | condary Chilled Water Pun | np(Variable Speed Pump) | | | | | | | |
| S | SCP ₁ | 800(GPM) | 6.0(KG/cm ²) | 50(%) | 50(%) | | | | |
| \$ | SCP ₂ | 800(GPM) | $6.0(KG/cm^2)$ | 50(%) | 50(%) | | | | |
| (IV) Co | ondenser Water Pump(Cons | tant Speed Pump) | | | | | | | |
| | CWP_1 | 1000(GPM) | 4.0 KG/cm^2 | 100(%) | 100(%) | | | | |
| (| CWP ₂ | 1000(GPM) | 4.0 KG/cm^2 | 100(%) | 100(%) | | | | |
| (V) Co | ooling Tower Fan(Variable S | Speed Fan) | | | | | | | |
| | CTF ₁ | 2200(CMM) | - | 50(%) | 50(%) | | | | |
| (| CTF ₂ | 2200(CMM) | - | 50(%) | 50(%) | | | | |
| | Operating Condition of Chi | ller Plant (parameter) | | Setting Value | | | | | |
| (1) | C Chilled Water | | 1 (Kcal/Kg-°C) | | | | | | |
| (2) | ^{ΔT} Chilled Water | | 5 (℃) | | | | | | |
| (3) | C Condenser Water | | 1 (Kcal/Kg-°C) | | | | | | |
| (4) | ^{ΔT} Condenser Water | | 5 (℃) | | | | | | |
| (5) | C Air | | 0.24 (Kcal/Kg-°(| 0.24 (Kcal/Kg-°C) | | | | | |
| (6) | ΔT_{Air} | | 5 (℃) | | | | | | |
| (7) | P _{SCP,i} | | 2.3 (Kg/KG/cm ² | $2.3 	ext{ (Kg/KG/cm}^2 	ext{)}$ | | | | | |
| | $P_{SCP,i}$ | | 3.8 (Kg/KG/cm ² | 3.8 (Kg/KG/cm ²) | | | | | |
| (9) | $P_{CWP,i}$ | | 3.2 (Kg/KG/cm ² | 3.2 (Kg/KG/cm ²) | | | | | |
| | Q _{max} | | 600 (RT) | 600 (RT) | | | | | |
| (11 | | | Depend on outside | Depend on outside air condition(dry bulb temperature \ relative humidity | | | | | |

The area in red labeled "number 6" records the optimal results, including the number of calculations, number of chillers that began operation, and the operating load of each chiller plant component.

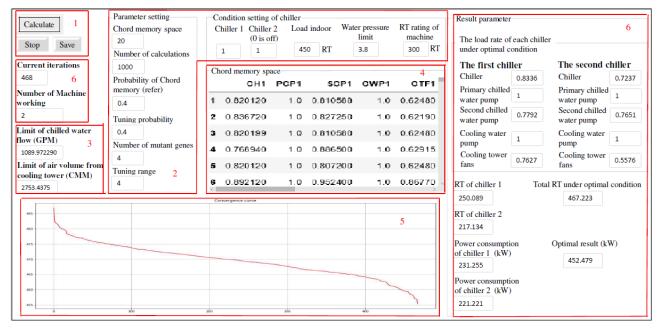


Figure 1. Operating interface of the decision analysis model for chiller plant energy optimization

4.3 Comparative analysis of efficiency (benefits)

Tables 2 and 3 present the power consumption and operating load of each component and the number of iterations calculated and performed using SA, GA, PSO, and HS when Q = 180 RT (30% total load), 210 RT(35% total load), 240 RT (40% total load), 270 RT (45% total load), 300 RT (50% total load), 330 RT (55% total load), 360 RT (60% total load), 390 RT (65% total load), 420 RT (70% total load), 450 RT (75% total load)

load), and 480 RT (80% total load). The number of iterations denotes the solution convergence speed of each algorithm. A high number of iterations indicates short convergence time; that is, the optimal solution can be determined within a short time frame. According to the tables, HS demonstrated more satisfactory energy-saving performance and calculation speed when compared with SA, GA, and PSO. Although HS had more power than PSO when Q=420, 450, and 480 RT, its calculation required fewer iterations. Therefore, HS was determined to be a cost-effective method that is advantageous for optimizing chiller plant operation. This study thus adopted HS as the problem-solving method for the decision analysis model for chiller plant energy optimization.

Table 2. Comparison of the solution results (power) of the four algorithms

| | RT | CH1 | PCP1 | SCP1 | CWP1 | CTF1 | CH2 | PCP2 | SCP2 | CWP2 | CTF2 | Power(Kw) | Saving(Kw) | Saving(%) |
|----------------|------------------|------------------|--------------------|------------------|--------------------|------------------|------------------|--------------------|------------------|--------------------|------------------|----------------------|--------------|-----------|
| Original | 180.00 | 0.00% | 0.00% | 86.67% | 0.00% | 86.67% | 60.00% | 100.00% | 86.67% | 100.00% | 86.67% | 239.0500 | - | - |
| SA | 180.14 | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 60.05% | 100.00% | 75.09% | 100.00% | 56.40% | 215.5851 | 23.4649 | 9.8159 |
| GA | 180.01 | 0.00% | 0.00% | 50.02% | 0.00% | 50.03% | 60.00% | 100.00% | 73.34% | 100.00% | 50.04% | 214.3441 | 24.7059 | 10.3350 |
| PSO | 180.00 | 0.00% | 0.00% | 50.00% | 0.00% | 50.09% | 60.00% | 100.00% | 73.34% | 100.00% | 50.02% | 214.3349 | 24.7151 | 10.3389 |
| HS | 180.00 | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 60.00% | 100.00% | 73.34% | 100.00% | 50.00% | 214.3326 | 24.7174 | 10.3399 |
| Original | 210.00 | 0.00% | 0.00% | 86.67% | 0.00% | 86.67% | 70.00% | 100.00% | 86.67% | 100.00% | 86.67% | 262.7500 | - | - |
| SA | 210.64 | 0.00% | 0.00% | 50.00% | 0.00% | 56.50% | 70.21% | 100.00% | 74.20% | 100.00% | 50.00% | 224.5322 | 38.2178 | 14.5453 |
| GA | 210.00 | 0.00% | 0.00% | 50.13% | 0.00% | 50.05% | 70.00% | 100.00% | 73.34% | 100.00% | 50.05% | 223.6317 | 39.1183 | 14.8880 |
| PSO | 210.01 | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 70.00% | 100.00% | 73.35% | 100.00% | 50.01% | 223.6105 | 39.1395 | 14.8961 |
| HS | 210.00 | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 70.00% | 100.00% | 73.34% | 100.00% | 50.00% | 223.6056 | 39.1444 | 14.8979 |
| Original | 240.00 | 40.00% | 100.00% | 90.00% | 100.00% | 90.00% | 40.00% | 100.00% | 90.00% | 100.00% | 90.00% | 333.8500 | - | - |
| SA | 240.89 | 80.30% | 100.00% | 73.66% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 54.02% | 231.1503 | 102.6997 | 30.7622 |
| GA | 240.02 | 80.01% | 100.00% | 73.52% | 100.00% | 50.03% | 0.00% | 0.00% | 50.02% | 0.00% | 50.04% | 230.6743 | 103.1757 | 30.9048 |
| PSO | 240.00 | 80.00% | 100.00% | 73.52% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.02% | 230.6653 | 103.1847 | 30.9075 |
| HS | 240.00 | 80.00% | 100.00% | 73.52% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 230.6608 | 103.1892 | 30.9088 |
| Original | 270.00 | 45.00% | 100.00% | 90.00% | 100.00% | 90.00% | 45.00% | 100.00% | 90.00% | 100.00% | 90.00% | 357.2500 | - | - |
| SA | 270.04 | 90.01% | 100.00% | 74.15% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 51.49% | 244.5667 | 112.6833 | 31.5419 |
| GA | 270.01 | 90.00% | 100.00% | 73.51% | 100.00% | 50.16% | 0.00% | 0.00% | 50.00% | 0.00% | 50.16% | 244.2063 | 113.0437 | 31.6427 |
| PSO | 270.00 | 90.00% | 100.00% | 73.52% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.07% | 244.1917 | 113.0583 | 31.6468 |
| HS | 270.00 | 90.00% | 100.00% | 73.52% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 244.1853 | 113.0647 | 31.6486 |
| Original | 300.00 | 50.00% | 100.00% | 90.00% | 100.00% | 90.00% | 50.00% | 100.00% | 90.00% | 100.00% | 90.00% | 380.6500 | - | |
| SA | 300.00 | 100.00% | 100.00% | 73.78% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 275.4246 | 105.2254 | 27.6436 |
| GA | 300.07 | 40.01% | 100.00% | 73.51% | 100.00% | 50.05% | 60.02% | 100.00% | 73.35% | 100.00% | 50.01% | 368.0908 | 12.5592 | 3.2994 |
| PSO | 300.00 | 100.00% | 100.00% | 73.51% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.41% | 275.3174 | 105.3326 | 27.6718 |
| HS | 300.00 | 100.00% | 100.00% | 73.52% | 100.00% | 50.00% | 0.00% | 0.00% | 50.00% | 0.00% | 50.00% | 275.3008 | 105.3492 | 27.6761 |
| Original | 330.00 | 55.00% | 100.00% | 93.33% | 100.00% | 93.33% | 55.00% | 100.00% | 93.33% | 100.00% | 93.33% | 411.3800 | 103.5452 | 27.0701 |
| SA | 334.97 | 40.00% | 100.00% | 75.37% | 100.00% | 67.42% | 71.66% | 100.00% | 73.38% | 100.00% | 55.01% | 381.1257 | 30.2543 | 7.3543 |
| GA | 330.00 | 40.00% | 100.00% | 73.51% | 100.00% | 50.03% | 70.00% | 100.00% | 73.42% | 100.00% | 50.05% | 377.3540 | 34.0260 | 8.2712 |
| PSO | 330.00 | 40.00% | 100.00% | 73.52% | 100.00% | 50.00% | 70.00% | 100.00% | 73.34% | 100.00% | 50.22% | 377.3229 | 34.0571 | 8.2787 |
| HS | 330.00 | 40.00% | 100.00% | 73.52% | 100.00% | 50.00% | 70.00% | 100.00% | 73.34% | 100.00% | 50.00% | 377.3113 | 34.0687 | 8.2816 |
| Original | 360.00 | 60.00% | 100.00% | 93.33% | 100.00% | 93.33% | 60.00% | 100.00% | 93.33% | 100.00% | 93.33% | 434.7800 | - | 0.2010 |
| SA | 361.87 | 40.00% | 100.00% | 76.07% | 100.00% | 62.25% | 80.62% | 100.00% | 74.61% | 100.00% | 50.00% | 391.1307 | 43.6493 | 10.0394 |
| GA | 360.01 | 40.00% | 100.00% | 73.51% | 100.00% | 50.02% | 80.00% | 100.00% | 73.36% | 100.00% | 50.05% | 387.7723 | 47.0077 | 10.8118 |
| PSO | 360.00 | 40.00% | 100.00% | 73.51% | 100.00% | 50.00% | 80.00% | 100.00% | 73.34% | 100.00% | 50.08% | 387.7642 | 47.0158 | 10.8137 |
| HS | 360.00 | 40.00% | 100.00% | 73.51% | 100.00% | 50.00% | 80.00% | 100.00% | 73.34% | 100.00% | 50.00% | 387.7596 | 47.0204 | 10.8148 |
| Original | 390.00 | 65.00% | 100.00% | 93.33% | 100.00% | 93.33% | 65.00% | 100.00% | 93.33% | 100.00% | 93.33% | 458.1800 | -77.0204 | 10.0140 |
| SA | 390.42 | 40.00% | 100.00% | 74.93% | 100.00% | 60.46% | 90.14% | 100.00% | 73.58% | 100.00% | 74.08% | 407.9981 | 50.1819 | 10.9524 |
| GA | 390.01 | 40.00% | 100.00% | 73.59% | 100.00% | 56.16% | 90.00% | 100.00% | 73.34% | 100.00% | 51.79% | 404.8732 | 53.3068 | 11.6345 |
| PSO | 390.00 | 40.00% | 100.00% | 73.52% | 100.00% | 54.34% | 90.00% | 100.00% | 73.34% | 100.00% | 53.61% | 404.8197 | 53.3603 | 11.6461 |
| HS | 390.00 | 40.00% | 100.00% | 73.52% | 100.00% | 54.72% | 90.00% | 100.00% | 73.34% | 100.00% | 53.21% | 404.8157 | 53.3643 | 11.6470 |
| Original | 420.00 | 70.00% | 100.00% | 93.33% | 100.00% | 93.33% | 70.00% | 100.00% | 93.33% | 100.00% | 93.33% | 481.5800 | - 33.30 13 | 11.0470 |
| SA | 420.00 | 40.00% | 100.00% | 74.01% | 100.00% | 75.95% | 100.00% | 100.00% | 76.55% | 100.00% | 54.49% | 436.7230 | 44.8570 | 9.3145 |
| GA | 420.00 | 84.83% | 100.00% | 73.63% | 100.00% | 57.31% | 55.17% | 100.00% | 73.36% | 100.00% | 58.95% | 431.8244 | 49.7556 | 10.3317 |
| PSO | 420.01 | 84.64% | 100.00% | 73.52% | 100.00% | 58.03% | 55.36% | 100.00% | 73.34% | 100.00% | 58.18% | 431.7623 | 49.8177 | 10.3446 |
| HS | 420.00 | 84.34% | 100.00% | 73.52% | 100.00% | 57.27% | 55.66% | 100.00% | 73.34% | 100.00% | 58.99% | 431.7726 | 49.8074 | 10.3440 |
| | 450.00 | 75.00% | | 93.33% | | 93.33% | 75.00% | | 93.33% | 100.00% | 93.33% | 504.9800 | 75.0074 | 10.3423 |
| Original SA | 454.55 | 86.08% | | 75.18% | | 80.58% | 65.44% | 100.00% | 77.59% | 100.00% | 50.89% | 449.2070 | 55.7730 | 11.0446 |
| GA | 450.01 | 81.35% | | 77.60% | | 63.98% | 68.65% | 100.00% | 73.57% | 100.00% | 60.54% | 444.3261 | 60.6539 | 12.0111 |
| PSO | 450.01 | 81.00% | | 77.79% | | 62.32% | 69.01% | | 73.35% | 100.00% | 62.18% | 444.3008 | 60.6792 | 12.0111 |
| HS | 450.01 | 81.00% | 100.00% | 77.75% | | 61.11% | 68.98% | 100.00% | 73.40% | 100.00% | 63.38% | 444.3011 | 60.6792 | 12.0162 |
| | | | | | | | | | | | | | | - |
| Original SA | 480.00 482.28 | 80.00% 82.53% | 100.00% 100.00% | 93.33% 82.69% | 100.00% 100.00% | 93.33% 83.63% | 80.00% 78.23% | 100.00% 100.00% | 93.33% 80.23% | 100.00% 100.00% | 93.33% 50.00% | 528.3800 463.1750 | - 65.2050 | 12.3406 |
| GA | 482.28 | 82.53% | | 82.69% | | 66.98% | 77.37% | 100.00% | 73.39% | 100.00% | 65.84% | 460.0867 | 68.2933 | 12.3406 |
| PSO | 480.04 | | | | | | 76.67% | | | 100.00% | 66.58% | 460.0867 | | |
| | | 83.33% | | 86.64% | | 66.22% | | | 73.46% | | | | 68.3458 | 12.9350 |
| HS | 480.00 | 83.44% | 100.00% | 83.44% | 100.00% | 66.73% | 76.56% | 100.00% | 73.52% | 100.00% | 66.11% | 460.0464 | 68.3336 | 12.9327 |

Table 3. Comparison of number of iterations of the four algorithms

| | 180(RT) | 210(RT) | 240(RT) | 270(RT) | 300(RT) | 330(RT) | 360(RT) | 390(RT) | 420(RT) | 450(RT) | 480(RT) | Total iterations |
|-----|---------|---------|---------|---------|---------|---------|---------|---------|---------|---------|---------|------------------|
| SA | 202 | 199 | 197 | 191 | 188 | 190 | 198 | 168 | 170 | 161 | 201 | 2065 |
| GA | 172 | 161 | 201 | 108 | 109 | 222 | 118 | 99 | 98 | 128 | 210 | 1626 |
| PSO | 162 | 155 | 190 | 128 | 110 | 162 | 109 | 141 | 129 | 138 | 142 | 1566 |
| HS | 143 | 125 | 180 | 128 | 109 | 158 | 107 | 140 | 121 | 130 | 131 | 1472 |

5. Conclusion

This study used the developed operation optimization decision analysis (support) system to determine the optimal operation status (operating load) of the chiller plant component equipment and thus obtain the optimal operation energy consumption when no existing equipment was changed and the level of comfort provided by the AC system was not influenced. After comparing the solution quality and convergence time of the four algorithms, this study discovered that HS exhibited the most satisfactory performance (efficiency) and was the easiest method to understand and implement.

6. Acknowledgment

This study was funded by the Ministry of Science and Technology (Project NSC-102-2218-E-492-003), and we greatly appreciate this support.

7. References

- [1]. Chang, Y.C., Optimal Chiller Loading by Evolution Strategy for Saving Energy, Energy and Buildings, 39, pp.437-444, 2007.
- [2]. C.W. Chen, C.W. Lee and C.Y. Chen, TO ENHANCE THE ENERGY EFFICIENCY OF CHILLER PLANTS WITH SYSTEM OPTIMIZATION THEORY, ENERGY & ENVIRONMENT, Vol. 21, No. 5, pp.409-424, 2010.
- [3]. Thomas B. Hartman, All-Variable Speed Centrifugal Chiller Plants, ASHRAE Journal, September 2001, pp.43-51, 2001.
- [4]. Yu, F.W. and K.T. Chan, Modeling of the coefficient of performance of an air-cooled screw chiller with variable speed condenser fans, Building and Environment, Vol. 41, pp.407-417, 2006.
- [5]. W.L. Winston, Operations Research: Applications and Algorithms, Duxbury Publisher, U.S.A., 1987.
- [6]. R.J. Dossat, Principles of Refrigeration, Wiley, U.S.A., 1991.